



## Inverse design of rotating metadvice for adaptive thermal cloaking

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### ABSTRACT

Thermal metamaterials have been extensively studied due to their extraordinary properties beyond natural materials and offered great flexibilities to tune heat flow for desired thermal functionalities, like thermal cloaking, concentrating, rotating, etc. Whereas, the thermal properties of thermal metamaterials are usually fixed once the configuration and the constituent materials are designed and fabricated. For instance, the thermal cloaking effect may be deteriorated when background changes, which limits its practical application significantly. By deducing the effective thermal conductivities of rotating objects, we propose an adaptive thermal cloaking metadvice that is composed by three rotating layers with different roles. The joint effect of three rotating layers makes the effective thermal conductivity a real number on the reciprocity line for feasible implementation. When background changes, we only need change the angular velocities rather than change the configuration or the constituent materials to restore the cloaking effect, which is much more convenient and real-time for practical applications. The underlying physics of the rotating thermal cloak is discussed to identify the key parameters and upper and lower limits of the effective thermal conductivity for further improving the cloaking effect. The present study can trigger more rotating metadvice for novel applications beyond thermal cloaking.

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### 1. Introduction

Thermal metamaterials have been attracting great interest since they have exhibited extraordinary properties beyond natural materials and offered great advantages for tuning heat flow almost at will. Based on thermal metamaterials, numerous thermal metadvice have been proposed for diverse functionalities, such as thermal cloak, thermal concentrator, thermal rotator, thermal camouflage, thermal illusion, etc. [1-25]. However, once the configuration and the constituent materials of the thermal metadvice are designed, the corresponding functionality can only be achieved at certain conditions [26-33]. In other words, most thermal metamaterials are usually unable to dynamically adapt to the change of environment. Even by changing the configuration or the constituent materials, the effective material parameters can only be switched between a finite number of discrete values, disabling the continuous regulation of the effective materials and the flexibility of these thermal metadvice. Let us take thermal cloak for an example. The methods of designing thermal cloak are mainly based on transformation thermotics [1-3] and scattering cancella-

tion techniques [4]. Both methods require the thermal conductivity tensor of the cloak meets certain conditions to achieve thermal cloaking effect. In transformation thermotics,  $\kappa' = \text{diag}(\frac{r-a}{r}, \frac{r}{r-a})\kappa_b$ , where  $a$  is the inner radius of the annular cloak in cylindrical coordinate, and  $\kappa_b$  is the thermal conductivity of the background; while in scattering cancellation, for a two-dimensional bilayer thermal cloak [4],  $\kappa'_2=0, \kappa'_3 = ((2c^3 + b^3)/(2(c^3 - b^3)))\kappa_b$ , where  $b$  and  $c$  are the radii of the outer layer. It is seen that the thermal conductivity of the cloak  $\kappa'$  is related to the thermal conductivity of the background  $\kappa_b$ , which means that the thermal cloaking effect cannot be maintained when the background changes. To restore the cloaking effect in the changing background, the material configuration or the constituent materials must be changed accordingly.

Recently, thermal convection [34-40] has garnered increasing attention as it can be used to continuously control the effective thermal conductivity by introducing velocity as another degree of freedom. But convection will break the Onsager reciprocity that exists in macroscopic heat conduction [34,35,41] and resulting asymmetric effective thermal conductivity tensor will distort the temperature field around the rotating object to some extent. Li et al. proposed that the effective thermal conductivity of the rotating object is usually a complex number due to the break of the reciprocity [35]. To restore the reciprocity and achieve the desired solid-like effect, they make the effective complex thermal conduc-

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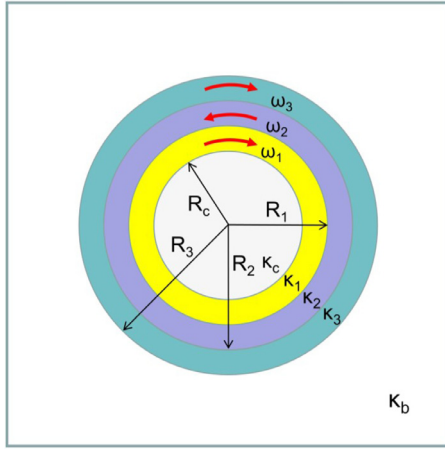


Fig. 1. Schematic diagram of the adaptive cloak.

tivity as a pure real number by reasonably adjusting the angular velocities [34].

In this work, we design a multi-layer adaptive thermal cloak that can adapt to the change of background and maintain the cloaking effect simultaneously. We firstly deduced the formula of the effective thermal conductivity of the three layers as we found that adaptive thermal cloaking can be achieved by three rotating layers. The inner rotating layer is used to maintain the cloaking region isothermal. The remaining two layers are designed to make the effective thermal conductivity as a real number which equals to the thermal conductivity of the background so that isotherms are parallel for ideal thermal cloaking effect. Finite element simulations were conducted to validate the formula and the adaptive thermal cloaking effect, and related discussions were presented to explain the underlying mechanisms. The present study is expected to trigger more rotating metadevices for novel applications beyond thermal cloaking.

function, and Eq. (1) can be solved analytically. Note that the temperature usually varies slowly, the  $G(\theta)$  can be given as the function of  $\theta$  as  $G(\theta) = e^{i\theta}$ . The temperature field needs to meet the following equation:

$$T(r, \theta) = T_0 + \frac{1}{2} [F(r) \cdot e^{i\theta} + \overline{F(r)} \cdot e^{-i\theta}] \quad (2)$$

where the overline denotes conjugation in complex number and  $T_0$  is a constant. Substituting Eq. (2) into Eq. (1), the general solutions of  $F(r)$  can be described as:

$$\begin{aligned} F_b(r) &= Ar + z_b r^{-1}, r \geq R_3 \\ F_3(r) &= z_5 M(\omega_3, r) + z_6 N(\omega_3, r), R_2 \leq r \leq R_3 \\ F_2(r) &= z_3 M(\omega_2, r) + z_4 N(\omega_2, r), R_1 \leq r \leq R_2 \\ F_1(r) &= z_1 M(\omega_1, r) + z_2 N(\omega_1, r), R_c \leq r \leq R_1 \\ F_c(r) &= z_c r, r \leq R_c \end{aligned} \quad (3)$$

where the coefficients  $z_c, z_b, z_1 - z_6$  are determined by the matching conditions of temperature and heat fluxes.  $M(\omega, r)$  and  $N(\omega, r)$  are:

$$\begin{aligned} M(\omega, r) &= \begin{cases} I_1(\sqrt{i\omega/D}r) & \omega \neq 0 \\ r & \omega = 0 \end{cases} \\ N(\omega, r) &= \begin{cases} K_1(\sqrt{i\omega/D}r) & \omega \neq 0 \\ 1/r & \omega = 0 \end{cases} \end{aligned} \quad (4)$$

where  $I_1(x)$  and  $K_1(x)$  are the first-order modified Bessel functions of the first and second kind.  $D$  is the diffusivity of the region rotating at  $\omega$ . Combining Eqs. (1-4), we can obtain the effective thermal conductivity of the multilayer rotating object:

$$\kappa_{c \sim n}^* = \kappa_n \frac{\kappa_n s_n - \kappa_{c \sim (n-1)}^* r_n}{\kappa_n q_n - \kappa_{c \sim (n-1)}^* p_n} \quad (5)$$

Therefore, the effective thermal conductivity of this three-layer object can be deduced by iterating Eq. (5). For  $n = 1$ ,  $\kappa_{c \sim 1}^* = \kappa_s \frac{\kappa_s s_1 - \kappa_{c \sim 0}^* r_1}{\kappa_s q_1 - \kappa_{c \sim 0}^* p_1}$ , where  $\kappa_{c \sim 0}^*$  is  $\kappa_c$ . For  $n = 2$ ,  $\kappa_{c \sim 2}^* = \kappa_s \frac{\kappa_s s_2 - \kappa_{c \sim 1}^* r_2}{\kappa_s q_2 - \kappa_{c \sim 1}^* p_2}$ , where  $\kappa_{c \sim 1}^*$  has already been deduced by using Eq. (5). Similarly,  $\kappa_{c \sim 3}^*$  can be obtained as

$$\kappa_{c \sim 3}^* = \kappa_s \frac{\kappa_s (q_1 q_2 s_3 - s_1 p_2 s_3 - q_1 s_2 r_3 + s_1 r_2 r_3) + \kappa_c (r_1 p_2 s_3 - p_1 q_2 s_3 + p_1 s_2 r_3 - r_1 r_2 r_3)}{\kappa_s (q_1 q_2 q_3 - s_1 p_2 q_3 - q_1 s_2 p_3 + s_1 r_2 p_3) + \kappa_c (r_1 p_2 q_3 - p_1 q_2 q_3 + p_1 s_2 p_3 - r_1 r_2 p_3)} \quad (6)$$

## 2. Theoretical analysis

Considering a two-dimensional cylindrical system, a temperature gradient is launched by maintaining two sides of a square background ( $L \times L$ ) with thermal conductivity  $\kappa_b$  at constant temperatures  $T_R$  and  $T_L$ . The upper and lower boundaries are thermally insulated. The cylindrical cloak, which consists of three concentric annuluses, as shown in Fig. 1, is used to cloak the center object. The thermal conductivities and radii are illustrated in Fig. 1. Diffusivities of three ring layers are  $D_1, D_2$  and  $D_3$ , respectively. The circular region is static while the angular velocities of three ring layers are  $\omega_1, \omega_2$  and  $\omega_3$ , respectively. The first layer rotates clockwise, the second layer rotates counterclockwise and the third layer rotates clockwise. The governing equations for the temperature fields  $T_c$  in  $r \leq R_c$ ,  $T_1$  in  $R_c \leq r \leq R_1$ ,  $T_2$  in  $R_1 \leq r \leq R_2$ ,  $T_3$  in  $R_2 \leq r \leq R_3$ ,  $T_b$  in  $r \geq R_3$  are

$$\begin{aligned} \frac{\partial^2 T_{b,c}}{\partial r^2} + \frac{1}{r} \frac{\partial T_{b,c}}{\partial r} + \frac{1}{r^2} \frac{\partial^2 T_{b,c}}{\partial \theta^2} &= 0 \quad 0 \leq r \leq R_c, r \geq R_3 \\ \frac{\partial^2 T_{1,2,3}}{\partial r^2} + \frac{1}{r} \frac{\partial T_{1,2,3}}{\partial r} + \frac{1}{r^2} \frac{\partial^2 T_{1,2,3}}{\partial \theta^2} &= \frac{\omega_{1,2,3}}{D_{1,2,3}} \frac{\partial T_{1,2,3}}{\partial \theta} \quad R_c \leq r \leq R_3 \end{aligned} \quad (1)$$

By variable separation method, the temperature profile could be explicitly written as  $T(r, \theta) = F(r) \cdot G(\theta)$  where  $G(\theta)$  is a periodic

where  $\kappa_s$  is the thermal conductivity of three ring layers. Here we set the thermal conductivity and the diffusivity of three ring layers as the same, i.e.  $\kappa_s = \kappa_1 = \kappa_2 = \kappa_3, D_1 = D_2 = D_3$ . The four cross-products  $p_n, q_n, r_n, s_n$  can be determined as

$$\begin{bmatrix} p_n \\ q_n \\ r_n \\ s_n \end{bmatrix}^T = \begin{bmatrix} \det \begin{bmatrix} M(\omega_n, R_n) & N(\omega_n, R_n) \\ M(\omega_n, R_{n-1}) & N(\omega_n, R_{n-1}) \end{bmatrix} \\ \det \begin{bmatrix} M(\omega_n, R_n) & N(\omega_n, R_n) \\ M'(\omega_n, R_{n-1}) & N'(\omega_n, R_{n-1}) \end{bmatrix} \\ \det \begin{bmatrix} M'(\omega_n, R_n) & N'(\omega_n, R_n) \\ M(\omega_n, R_{n-1}) & N(\omega_n, R_{n-1}) \end{bmatrix} \\ \det \begin{bmatrix} M'(\omega_n, R_n) & N'(\omega_n, R_n) \\ M'(\omega_n, R_{n-1}) & N'(\omega_n, R_{n-1}) \end{bmatrix} \end{bmatrix}^T \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & R_{n-1} & 0 & 0 \\ 0 & 0 & R_n & 0 \\ 0 & 0 & 0 & R_{n-1} R_n \end{bmatrix} \quad (7)$$

Here,  $R_0$  is  $R_c$ . It should be noted that  $M'(\omega, r)$  and  $N'(\omega, r)$  here only take derivatives of  $M(\omega, r)$  and  $N(\omega, r)$  to  $r$ . The effective thermal conductivity in Eq. (6) is independent of the external parameters ( $L, \kappa_b, T_R, T_L$ ) and only influenced by the structure parameters.

Based on convection, rotating allows continuous adjustment of the effective thermal conductivity over a wide range breaking the limit of Maxwell-Garnett mixing rule. However,  $\kappa_{c \sim 3}^*$  ( $\kappa_{c \sim 3}^*$  is mentioned as  $\kappa^*$  hereinafter for simplification) is a complex number because of the properties of the modified Bessel functions. The imaginary part of the effective thermal conductivity essentially comes from the broken reciprocity, resulting in that the temperature field around rotating objects will be distorted to some extent. Therefore, it is not generally applicable to the regulation requirements of conventional thermal metamaterials [28,39,42,43]. To eliminate the advection effect of the rotating layers, the angular velocity of each layer should be set reasonably to make the calculated effective thermal conductivity as a pure real number. The real-number effective thermal conductivity restores the reciprocity and makes rotating objects achieve a solid-like effect. Obviously, the direction of rotating should be opposite to cancel each other out so that the effective thermal conductivity can be a real number. With reasonable adjustment of the angular velocities, the effective thermal conductivity can be continuously changed to adapt the change in the environment while maintaining the thermal cloaking effect.

### 3. Results and discussions

Based on the effective thermal conductivity obtained above, we design an adaptive cloak as shown in Fig. 1. The whole simulation region is  $200 \times 200 \text{ mm}^2$ . Here we set  $\kappa_c = \kappa_s = \kappa_1 = \kappa_2 = \kappa_3 = 50 \text{ W m}^{-1} \text{ K}^{-1}$ ,  $D_1 = D_2 = D_3 = 13.3 \text{ mm}^2 \text{ s}^{-1}$ ,  $R_c = 0.03 \text{ m}$ ,  $R_1 = 0.04 \text{ m}$ ,  $R_2 = 0.05 \text{ m}$ ,  $R_3 = 0.06 \text{ m}$ . Ideal thermal cloaking effect requires (1) The temperature gradient vanishes in the center region where the object is placed and (2) The isotherms outside the cloak region are parallel. To achieve these requirements, a three-ring-layer structure is set up specifically. The first layer has a large angular velocity to eliminate the temperature gradient inside the center region. The angular velocities of the second and the third layers should be adjusted properly to make the effective thermal conductivity  $\kappa^*$  as a real number and equal to the thermal conductivity of the background  $\kappa_b$  so as to make the external isotherms parallel according to the scattering cancellation principle.

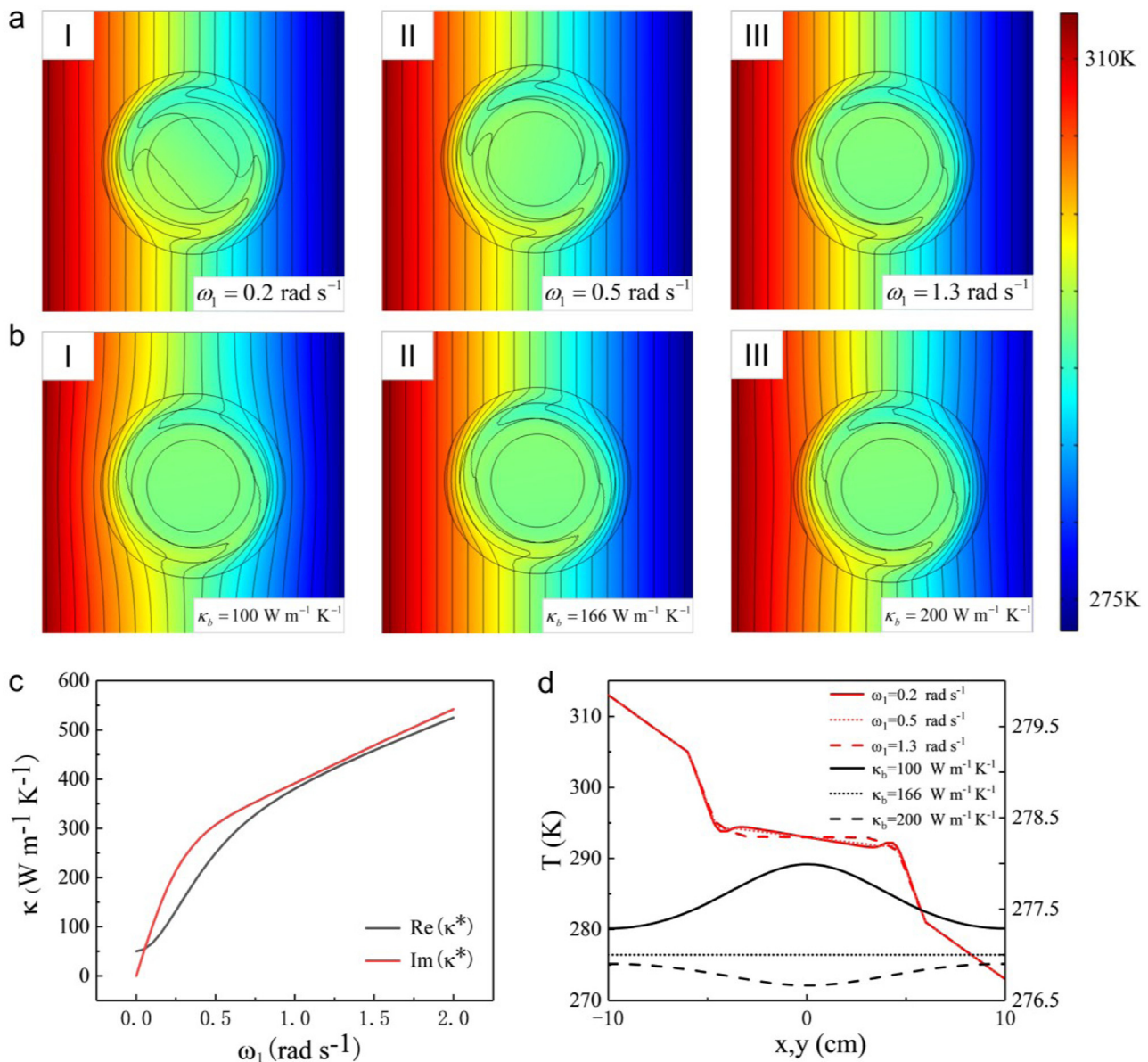
Fig. 2a illustrates the simulated temperature fields with varying angular velocities  $\omega_1$ . It is seen in Fig. 2a that when  $\omega_1$  is small (I,  $0.2 \text{ rad s}^{-1}$ ), the temperature inside the circular region is not uniform, while  $\omega_1$  is large enough (III,  $1.3 \text{ rad s}^{-1}$ ), the temperature gradient inside the circular region vanishes. Meanwhile, the isotherms outside the cloak are always vertical and uniform. Fig. 2b shows the simulated temperature fields with fixed angular velocities ( $\omega_1$ :  $1.3 \text{ rad s}^{-1}$ ,  $\omega_2$ :  $-0.5 \text{ rad s}^{-1}$ ,  $\omega_3$ :  $0.05688 \text{ rad s}^{-1}$ ) and varying background's thermal conductivities from 100, 166, to  $200 \text{ W m}^{-1} \text{ K}^{-1}$ . From Eq. (6), the effective thermal conductivity  $\kappa^*$  in this case is calculated as  $166 \text{ W m}^{-1} \text{ K}^{-1}$ . When  $\kappa^*$  is larger (I) or smaller (III) than  $\kappa_b$ , the external isotherms are not parallel and the thermal cloaking effect is deteriorated. Only when  $\kappa^*$  equals  $\kappa_b$ , the external isotherms are parallel, which meets the second criterion of thermal cloaking. Fig. 2c shows how the effective thermal conductivity of the circular region and first layer varies with different  $\omega_1$ . When  $\omega_1$  increases, the real part and the imaginary part of the effective thermal conductivity increase accordingly, which is why the first criterion of thermal cloaking effect can be achieved at large  $\omega_1$ . To quantify the thermal cloaking effect, temperature profiles along the horizontal cutline at  $y=0$  (red lines) and vertical cutline at  $x=8 \text{ cm}$  (black lines) are shown in Fig. 2d. The temperature gradient in the circular region decreases with the increasing of  $\omega_1$  and when  $\omega_1$  is  $1.3 \text{ rad s}^{-1}$ , the temperature gradient is almost zero, which validates the first criterion of thermal cloaking effect. When  $\kappa^*$  equals  $\kappa_b$ , the temperature profile is uniform as

well, validating the parallel isotherms of the thermal cloaking effect.

To sum up, the three-layer structure can be adjusted to achieve different effective thermal conductivities and thermal cloaking effect can be accomplished when: (1)  $\omega_1$  is large enough to make the temperature gradient vanish inside the cloaking region; (2)  $\omega_2$  and  $\omega_3$  are properly designed to make  $\kappa^*$  as a real number that equals to  $\kappa_b$ . For the two-layer rotating structure, the angular velocities of these two layers should be regulated properly to make the effective thermal conductivity the same as background's thermal conductivity. When the background changes, the cloaking effect will not be maintained if the angular velocity of the first layer is small in a certain case. While, for the three-layer scheme, the adaptability of the changing background has more freedom because three angular velocities can be adjusted. If more layers are used for adaptive cloak, like four layers, the adaptive cloaking effect can be achieved at the expense of more complicated structure and implementation.

So far, we have only demonstrated the static cloaking effect, which cannot be called as 'adaptive'. For this end, the angular velocities need to be calculated at different  $\kappa_b$  to meet those two criteria of thermal cloak. From Eq. (6), the effective thermal conductivity can be calculated by inputting the radius, diffusivity and angular velocity. In return, the angular velocities can also be inversely calculated by providing the radius, diffusivity and expected thermal conductivity. When background changes, the proper angular velocities can be calculated inversely to maintain the cloaking effect.

When the background ( $\kappa_b$ ) changes, we can calculate angular velocities inversely by Eq. (6), and then modify the angular velocities manually or automatically to maintain the cloaking effect. To make  $\kappa^*$  equal to  $\kappa_b$ ,  $\kappa^*$  should be a real number firstly to keep the reciprocity. For a certain  $\omega_1$ , the matching  $\omega_2$  and  $\omega_3$  are not unique, and so is the real-number effective thermal conductivity. As shown in Fig. 3a, the matching  $\omega_2$  and  $\omega_3$  can be plotted as  $\omega_1$ -dependent lines to make the effective thermal conductivity as a real number. Such lines are called as reciprocal lines. For a certain  $\omega_1$ , the matching  $\omega_3$  increases rapidly at first and then decreases gradually as  $\omega_2$  increases. For the same  $\omega_2$ , the larger  $\omega_1$  is, the smaller matching  $\omega_3$  is. The reason is that both of the first layer and the third layer rotate clockwise but the second layer rotates counterclockwise. If  $\omega_1$  is larger, the matching  $\omega_3$  can be smaller accordingly. It is seen that  $\omega_1$  matters for the matching  $\omega_3$  only at the beginning. This is because when  $\omega_2$  increases gradually, its influence on  $\kappa^*$  is much greater than that of  $\omega_1$ . We can also find that  $\omega_2$  and  $\omega_3$  differ by 1–2 orders of magnitude, which proves the outermost layer has the greatest influence on  $\kappa^*$ . When  $\omega_2$  changes in a large range,  $\omega_3$  only changes in a small range and the matching conditions can be achieved. After calculating the reciprocal line, the real-number  $\kappa^*$  can be calculated by using three angular velocities on the reciprocal line. As shown in Fig. 3b, with the increasing of  $\omega_2$ ,  $\kappa^*$  decreases at the beginning and then converges gradually. The reason for the initial decrease of  $\kappa^*$  is that the first layer and the second layer rotate reversely, and when  $\omega_2$  increases gradually from zero, the effect of clockwise rotation of the first layer will be compromised at the beginning. After completely canceling the influence of the first layer, the counterclockwise rotation of the second layer matters dominantly, so  $\kappa^*$  increases when  $\omega_2$  increases.  $\kappa^*$  converges eventually, which means there is an upper limit of  $\kappa^*$ . For different  $\omega_1$ , the influence is also only at the initial stage where  $\omega_1$  plays a major role. The smaller  $\omega_1$  is, the smaller  $\kappa^*$  is. According to Fig. 3b, the tunable range of  $\kappa^*$  is limited. Here, the regulatory range of  $\kappa^*$  is about  $100\text{--}250 \text{ W m}^{-1} \text{ K}^{-1}$ , which surprisingly validates that rotating makes metamaterials beyond the Maxwell-Garnett formula (All thermal conductivities here are  $50 \text{ W m}^{-1} \text{ K}^{-1}$ ). That is to say, the range of  $\kappa_b$  that the thermal cloak can



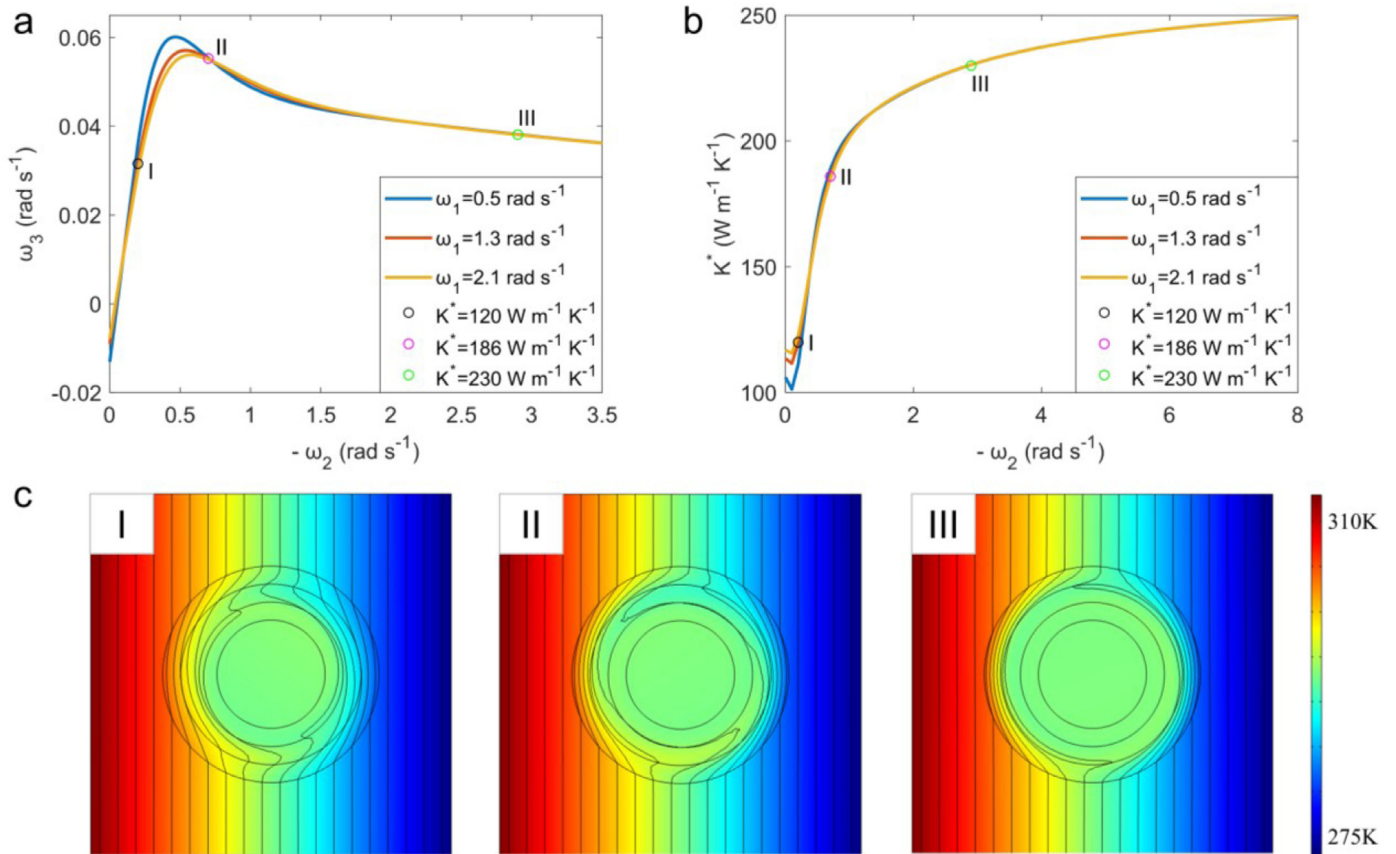
**Fig. 2.** Simulation validations of thermal cloaking effect with varying parameters. (a) Simulated temperature fields with varying  $\omega_1$  from 0.2, 0.5, to 1.3  $\text{rad s}^{-1}$ , respectively. (b) Simulated temperature fields with varying thermal conductivities of background from 100, 166, to 200  $\text{W m}^{-1} \text{K}^{-1}$  and fixed angular velocities ( $\omega_1: 1.3 \text{ rad s}^{-1}$ ,  $\omega_2: -0.5 \text{ rad s}^{-1}$  and  $\omega_3: 0.05688 \text{ rad s}^{-1}$ ). (c) The  $\omega_1$ -dependent real part and imaginary part of the effective thermal conductivity of circular region and the first layer. (d) Temperature profiles along cutlines of  $y=0$  (red lines) and  $x=8 \text{ cm}$  (black lines). (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

adapt to is within  $100\text{--}250 \text{ W m}^{-1} \text{K}^{-1}$ . Three  $\kappa_b$  ( $120 \text{ W m}^{-1} \text{K}^{-1}$ ,  $186 \text{ W m}^{-1} \text{K}^{-1}$  and  $230 \text{ W m}^{-1} \text{K}^{-1}$ , respectively) are selected to test the adaptive thermal cloaking effect. As shown in Fig. 3c, the thermal cloaking effects are all achieved ideally by changing  $\omega_2$  and  $\omega_3$ . These setting parameters of the three selected cloaks are marked in Figs. 3a and 3b. If  $\kappa_b$  changes within the tunable range of  $\kappa^*$ , we can find appropriate  $\omega_2$  and  $\omega_3$  to achieve good thermal cloaking effect even when  $\kappa_b$  changes continuously, which is almost impossible for traditional thermal cloak.

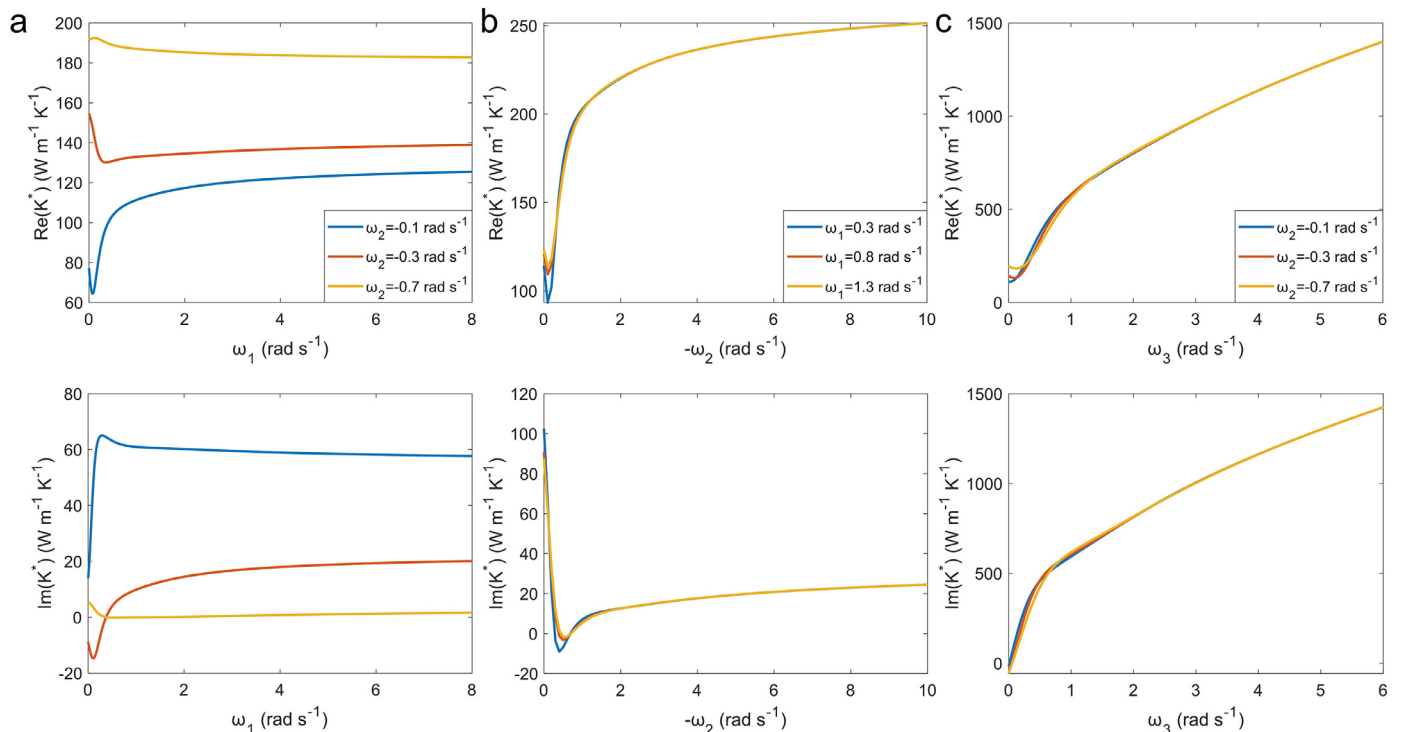
Next, to explore the physical significance of the three angular velocities, the influence of angular velocities on  $\kappa^*$  is further analyzed in Fig. 4. We discuss the real part and the imaginary part respectively. The real part and the imaginary part of  $\kappa^*$  are related to dissipation and propagation. In Fig. 4a, when  $\omega_2$  are  $-0.1 \text{ rad s}^{-1}$  and  $-0.3 \text{ rad s}^{-1}$ ,  $\text{Re}(\kappa^*)$  decreases at first and then increases and finally converges to a certain value. The reason why it decreases at

first is the offset effect by different rotating directions of the first and second layer. When  $\omega_2$  is  $0.7 \text{ rad s}^{-1}$ ,  $\text{Re}(\kappa^*)$  increases at first and then decreases slowly. While  $\text{Im}(\kappa^*)$  varies differently at different  $\omega_2$ .  $\omega_1$  has little influence on  $\kappa^*$ , where the regulatory range is about a few dozen. Similarly,  $\text{Re}(\kappa^*)$  and  $\text{Im}(\kappa^*)$  both decrease and then increase because of the offset effect when  $\omega_2$  increases. The regulatory range for different  $\omega_2$  is a few hundred, which is larger than that for  $\omega_1$ . For different  $\omega_3$ ,  $\text{Re}(\kappa^*)$  decreases and then increases due to the offset effect while  $\text{Im}(\kappa^*)$  keeps increasing. Rotating of the third layer has the strongest influence on the effective thermal conductivity. Different from the first two layers,  $\kappa^*$  does not converge to a certain value when  $\omega_3$  increases, and thus the influence of the outer layer is more significant on the effective thermal conductivity.

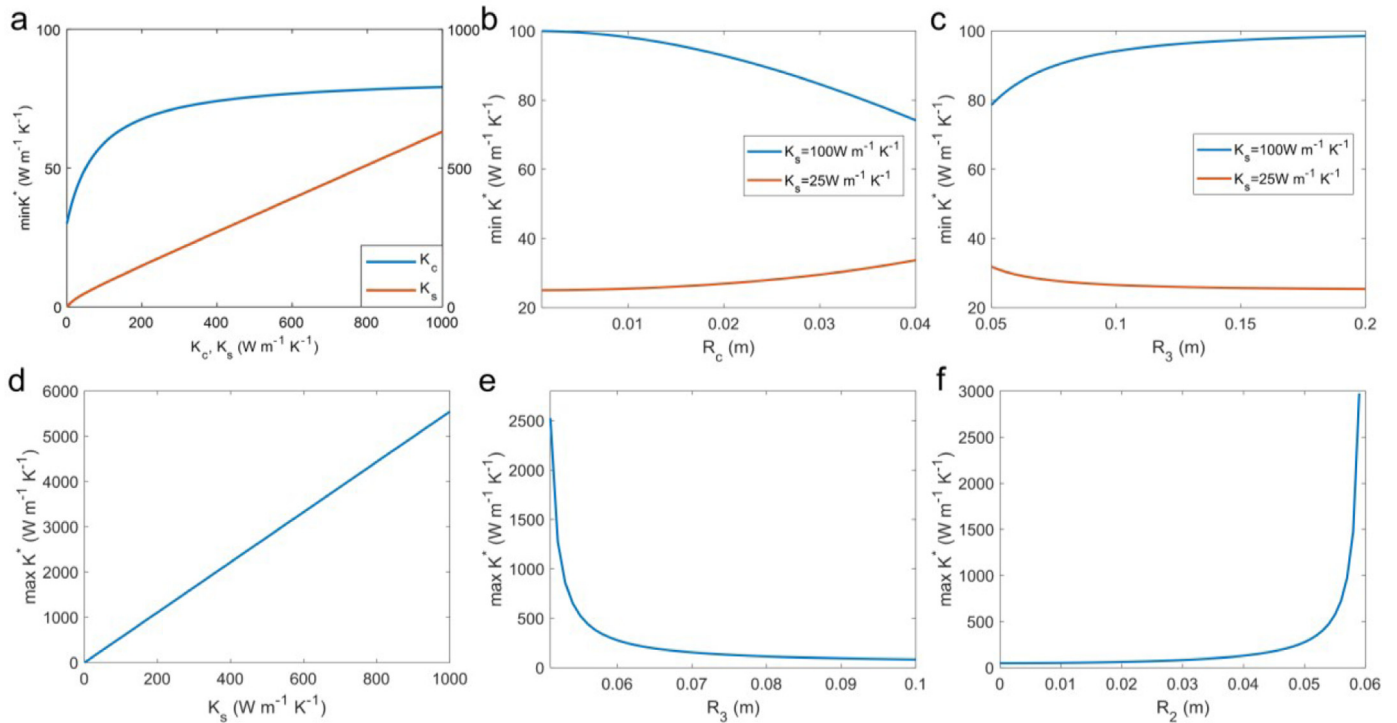
To make the thermal cloak have a wider range of regulation, we also explore the upper and lower limits of  $\kappa^*$ . Obviously,  $\kappa^*$  is



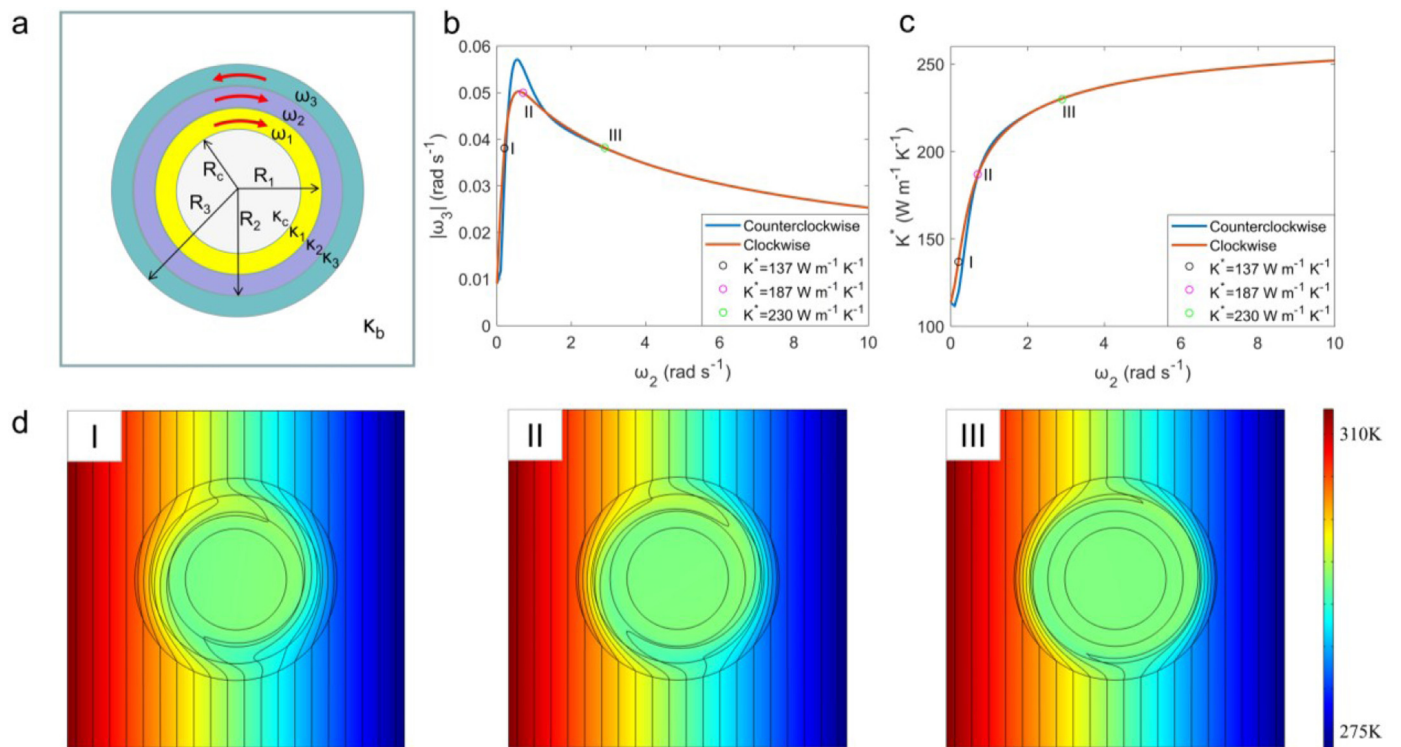
**Fig. 3.** Discussion on the reciprocal line of  $\kappa^*$  for the adaptive thermal cloak. (a) Reciprocal line for different  $\omega_1$ . (b) The real-number effective thermal conductivity on three reciprocal lines. (c) Simulations of the adaptive thermal cloak.  $\kappa_b$  for I-III are 120 W m<sup>-1</sup> K<sup>-1</sup>, 186 W m<sup>-1</sup> K<sup>-1</sup>, 230 W m<sup>-1</sup> K<sup>-1</sup>, respectively. Angular velocities are calculated respectively.  $\omega_1, \omega_2, \omega_3$  are 1.3 rad s<sup>-1</sup>, -0.2 rad s<sup>-1</sup>, 0.031588 rad s<sup>-1</sup> for I. 1.3 rad s<sup>-1</sup>, -0.7 rad s<sup>-1</sup>, 0.055290 rad s<sup>-1</sup> for II. 1.3 rad s<sup>-1</sup>, -2.9 rad s<sup>-1</sup>, 0.038097 rad s<sup>-1</sup> for III.



**Fig. 4.** The influence of three angular velocities on the real part and the imaginary part of  $\kappa^*$ . (a) The influence of  $\omega_1$  on  $\kappa^*$  at different  $\omega_2$  with fixed  $\omega_3$ (0.055290 rad s<sup>-1</sup>). (b) The influence of  $\omega_2$  on  $\kappa^*$  at different  $\omega_1$  with fixed  $\omega_3$ (0.055290 rad s<sup>-1</sup>). (c) The influence of  $\omega_3$  on  $\kappa^*$  at different  $\omega_2$  with fixed  $\omega_1$ (1.3 rad s<sup>-1</sup>).



**Fig. 5.** The influence of each parameter on  $\min \kappa^*$  and  $\max \kappa^*$ . (a-c) The influence of each parameter on  $\min \kappa^*$ . (d-f) The influence of each parameter on  $\max \kappa^*$ . For subfigures (a-f),  $\kappa_c = \kappa_3 = 50$  W m<sup>-1</sup> K<sup>-1</sup>,  $R_c = 0.03$  m,  $R_1 = 0.04$  m,  $R_2 = 0.05$  m,  $R_3 = 0.06$  m.



**Fig. 6.** The influence of reverse rotation in the second and the third layers. (a) Schematic diagram of reverse rotation in the second and the third layers. (b) Comparison of reciprocal lines when the second layer rotates counterclockwise (the third layer rotates clockwise) and clockwise (the third layer rotates counterclockwise). (c) Comparison of  $\kappa^*$  on the reciprocal lines. (d) Simulations of the adaptive cloak when  $\kappa_b$  changes.  $\kappa_b$  for I-III are 137 W m<sup>-1</sup> K<sup>-1</sup>, 187 W m<sup>-1</sup> K<sup>-1</sup>, 230 W m<sup>-1</sup> K<sup>-1</sup>. For I,  $\omega_1$ : 1.3 rad s<sup>-1</sup>,  $\omega_2$ : 0.2 rad s<sup>-1</sup>,  $\omega_3$ : -0.038103 rad s<sup>-1</sup>. For II,  $\omega_1$ : 1.3 rad s<sup>-1</sup>,  $\omega_2$ : 0.7 rad s<sup>-1</sup>,  $\omega_3$ : -0.049999 rad s<sup>-1</sup>. For III,  $\omega_1$ : 1.3 rad s<sup>-1</sup>,  $\omega_2$ : 2.9 rad s<sup>-1</sup>,  $\omega_3$ : -0.038182 rad s<sup>-1</sup>.

the smallest when three angular velocities are all zero. By using the properties of  $M(\omega, r)$  and  $N(\omega, r)$ , the lower limit of  $\kappa^*$  can be derived as

$$\min \kappa^* = \kappa_s \cdot \frac{\left(\frac{\kappa_c}{\kappa_s} - 1\right) \left(1 + \frac{R_c^2}{R_3^2}\right) + 2}{\left(\frac{\kappa_c}{\kappa_s} - 1\right) \left(1 - \frac{R_c^2}{R_3^2}\right) + 2}. \quad (8)$$

It is seen from Eq. (8) that the minimum value of  $\kappa^*$  is only related to parameters of the structure  $(\kappa_c, \kappa_s, R_c, R_3)$ . Fig. 5a-c illustrate the influence of each parameter on  $\min \kappa^*$ .  $\min \kappa^*$  increases with the increasing of  $\kappa_c$  and  $\kappa_s$ . The influence of  $R_c$  and  $R_3$  depends on the sign of  $(\kappa_c/\kappa_s - 1)$ . The upper limit of  $\kappa^*$  is taken when  $\omega_2 = \infty$ . The modified Bessel functions have the following properties:  $\lim_{x \rightarrow \infty} I_1'(x) = \lim_{x \rightarrow \infty} I_1(x)$ ,  $\lim_{x \rightarrow \infty} K_1'(x) = \lim_{x \rightarrow \infty} K_1(x)$ . When  $\omega_2 = \infty$ , the matching  $\omega_3$  on the reciprocal line is zero. Therefore, the formula of  $\max \kappa^*$  can be obtained as

$$\max \kappa^* = \kappa_s \cdot \frac{R_3^2 + R_2^2}{R_3^2 - R_2^2}. \quad (9)$$

Similarly,  $\max \kappa^*$  is only related to parameters of the structure  $(\kappa_s, R_2, R_3)$ . Fig. 5d-f illustrate the influence of each parameter on  $\max \kappa^*$ . When  $R_3$  is much larger than  $R_2$ ,  $\max \kappa^*$  approaches  $\kappa_s$ , which is reasonable in practice. When  $R_2$  approaches  $R_3$ ,  $\max \kappa^*$  can be infinite. Parameters can be chosen properly to get a satisfying range of  $\kappa^*$  by using Eqs. (8) and (9), which provide a basis for choosing parameters of the adaptive thermal cloak. For example, to obtain a wide range of regulation, best parameters can be calculated within the range of  $\max \kappa^*$  and  $\min \kappa^*$ . What's more, to get a higher (lower)  $\max \kappa^*$  ( $\min \kappa^*$ ), the required parameters in Eq. (8) or Eq. (9) can be adjusted accordingly. The upper and lower limits of more multilayered structures can be derived as well in a similar way.

Finally, we explore how the reciprocal line and  $\kappa^*$  on the reciprocal line change when the second and the third layers rotate reversely, as shown in Fig. 6a. Can the matching  $\omega_3$  for a certain  $\omega_2$  be smaller by changing the directions of  $\omega_2$  and  $\omega_3$  so that we can reduce the input of power? Or can we get a wider regulation range of  $\kappa^*$  for certain angular velocities by changing the directions of  $\omega_2$  and  $\omega_3$ ? From Fig. 6b, there is a difference of  $\omega_3$  at around  $\omega_2=1 \text{ rad s}^{-1}$  when the second and the third layers rotate reversely, which can reduce input of power to some extent. Fig. 6c shows that the influence of rotating reversely on  $\kappa^*$  is concentrated in small- $\omega_2$  region. The same effective thermal conductivity can be obtained from a smaller angular velocity after rotating reversely. Fig. 6d is the simulation of the adaptive reverse-cloak at different  $\kappa_b$ . In different background, the cloaking effect is still achieved well only by changing angular velocities. Parameters of three simulations are marked in Fig. 6b and c. Finally, some discussions about experimental validations are provided. The three layers can rotate with motors and gears on their bottom surfaces, referring to implementation of the two-layer structure [34]. Thermal conductive silicone grease can be used to fill the gaps between the object, the layers and background, which helps to eliminate the interface thermal resistance in the experiment.

#### 4. Conclusion

In this paper, we derive the effective thermal conductivity of the three-layer rotating metadvice and design a three-rotating-layer adaptive thermal cloak inversely. The inner rotating layer is used to maintain the cloaking region isothermal. The remaining two layers are designed to make the effective thermal conductivity as a real number which equals to the thermal conductivity of the background so that isotherms are parallel for adaptive thermal

cloaking. When background changes, we only need change the angular velocities rather than change the configuration or the constituent materials to restore the cloaking effect, which is much more convenient and real-time for practical applications. Both finite element simulations and subsequent analysis are carried out to explore the influence of each angular velocity on the effective thermal conductivity, the upper and lower limits of the effective thermal conductivity, and the influence of reverse rotation. Beyond thermal cloaking, more thermal functionalities and novel applications are expected to achieve by resorting to the rotating metadevices in the near future.

#### Declaration of Competing Interest

There are no conflicts of interest.

#### CRediT authorship contribution statement

**Zhan Zhu:** Investigation, Methodology, Writing - original draft. **Xuecheng Ren:** Investigation, Data curation. **Wei Sha:** Writing - review & editing. **Mi Xiao:** Writing - review & editing. **Run Hu:** Conceptualization, Supervision, Writing - review & editing. **Xiaobing Luo:** Writing - review & editing.

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